

# Structural optimization of an automotive wheel rim using the BGM method

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# INTRODUCTION



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In the *Automotive* sector, wheel rims play both a structural and stylistic role.

They significantly contribute to the vehicle's aesthetics, brand perception, and market positioning.

In discussions with Nissan, it emerged that the Automotive industry is driven by the ***Design First*** paradigm.

This approach imposes strict constraints on the design engineer.



# INTRODUCTION

- Real-world geometry provided by Nissan, subject to design constraints.
- Preliminary structural analysis performed using the FEM method
- Two-phase optimization:
  - Mesh morphing for mass reduction.
  - BGM to improve stress distribution.
- Activity carried out in Ansys Workbench, using Ansys Mechanical and RBF Morph.



# OBJECTIVES



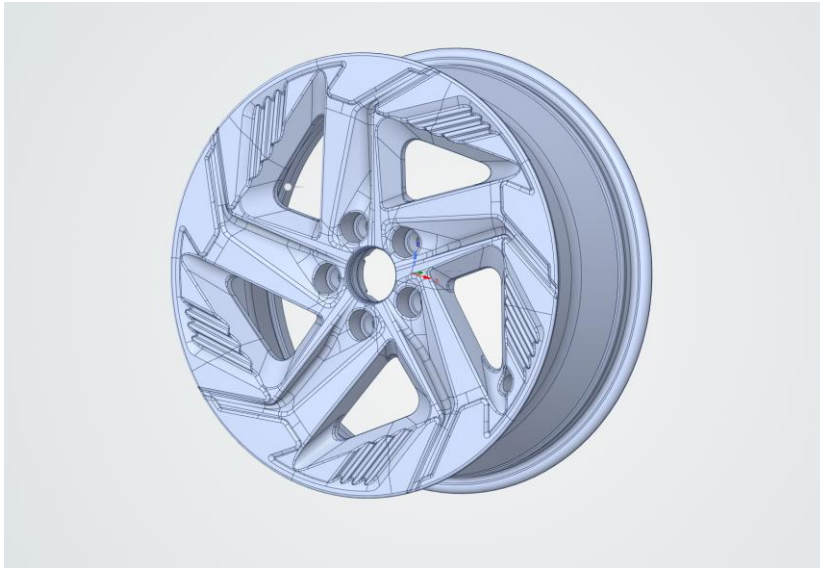
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- To optimize a real wheel rim provided by Nissan, while preserving its original design.
- Mass reduction through mesh morphing with stress control.
- Optimization of stress distribution using the Biological Growth Method (BGM).

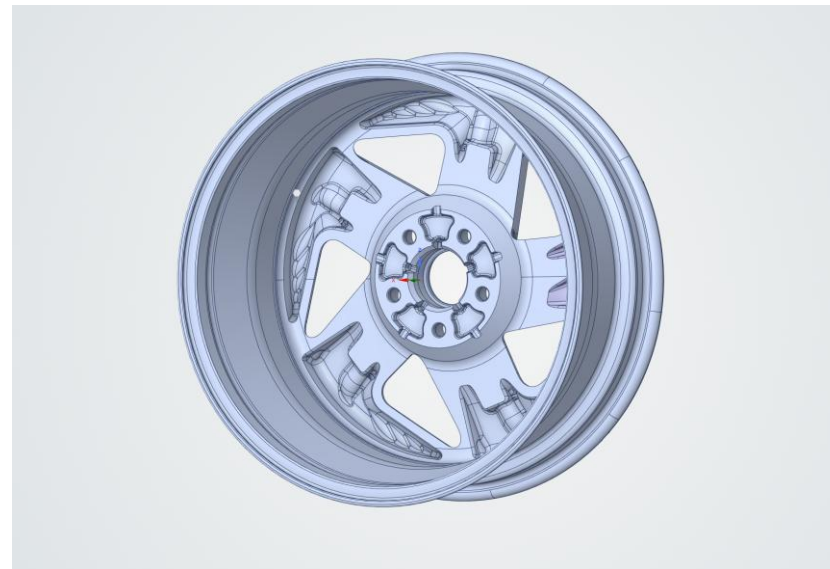
# CASE STUDY



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Parameters	Value
Mass	14,1 kg
Diameter	0,50 m
Density	2700 kg/m <sup>3</sup>
Young's Modulus	71 GPa
Yield Strength	190 MPa



- Five-spoke design
- *Material:* Aluminum Alloy AlSi7Mg0.3

# LOAD TESTS



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Nissan provided three load tests, internally developed by the company:

- **Rotary bending test:** simulates the lateral forces acting on the wheel during cornering;
- **Impact test:** simulates forces generated by road surface irregularities;
- **Drum durability test:** simulates the stresses experienced by the wheel during ground contact.

# STRUCTURAL ANALYSIS



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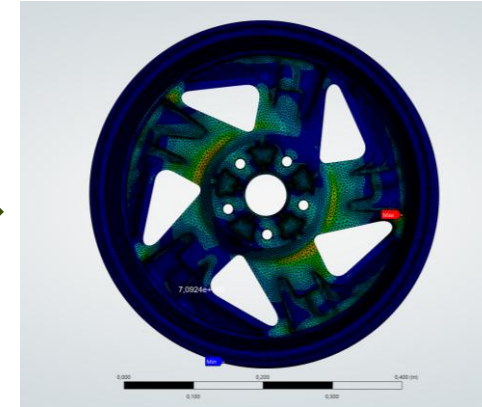
CAD



MESH



RESULTS



FEM

# STRUCTURAL ANALYSIS



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Load Test	Maximun Stress
Drum durability test	81,24 MPa
Rotay bending test	69,9 MPa
Impact Test	53,3 MPa



The drum durability test proves to be the most demanding.



$$k = \frac{\sigma_{max}}{\sigma_{allowable}} < 1$$



$$k = 0,43 < 1$$



Optimization is possible!

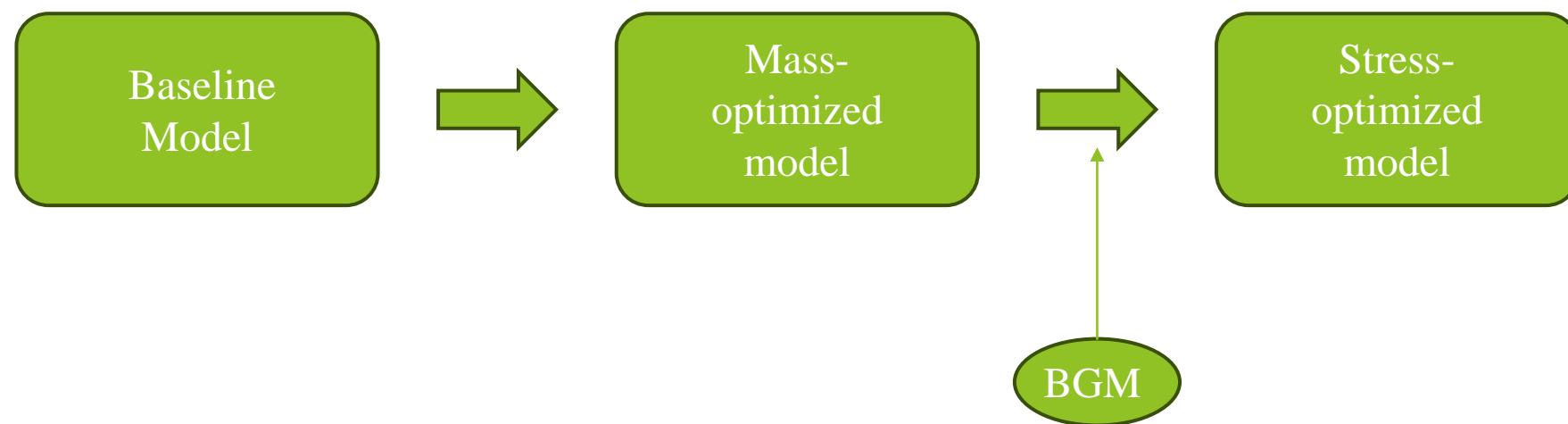




# OPTIMIZATION WORKFLOW



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# SHAPE OPTIMIZATION



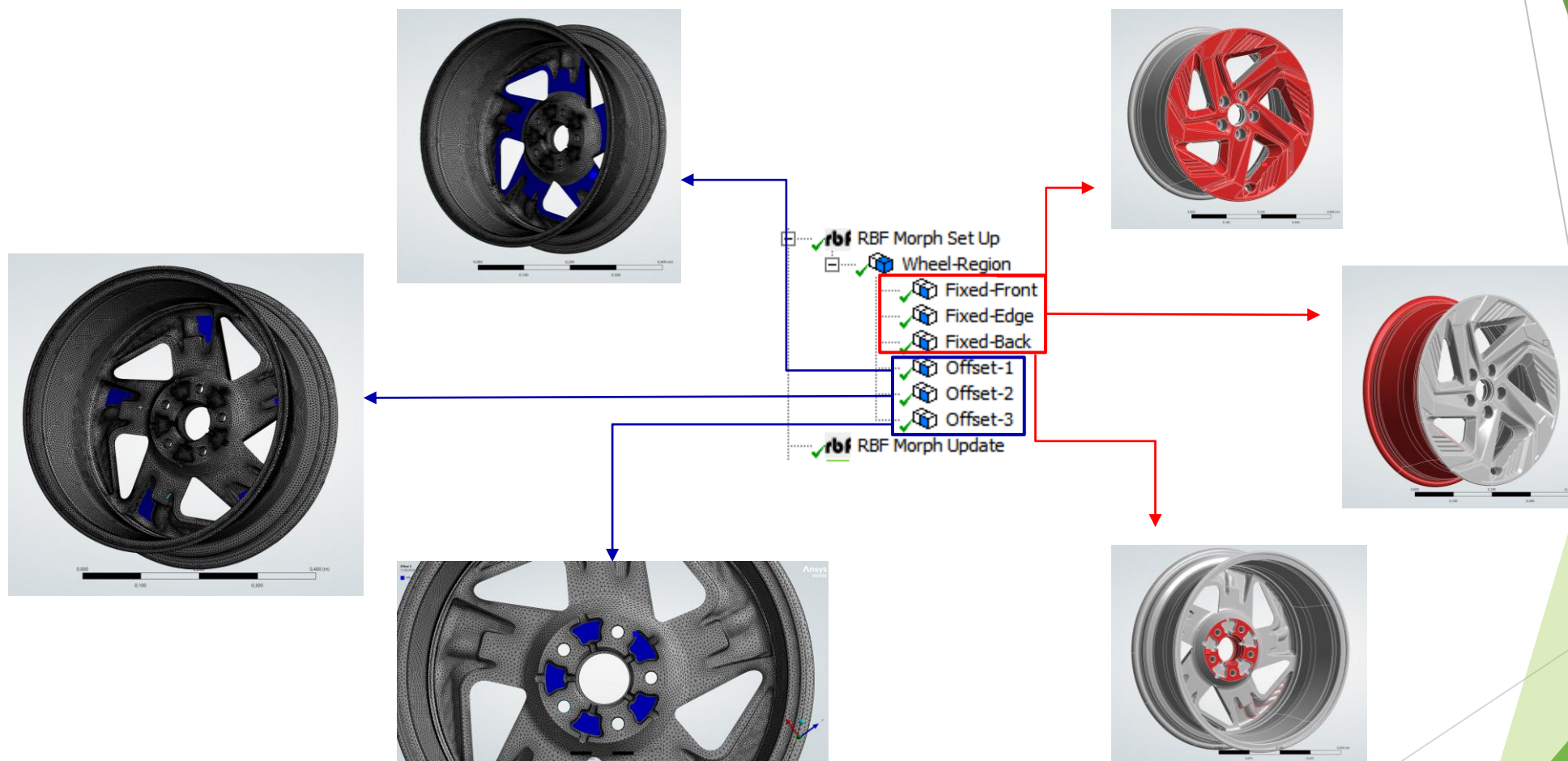
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- Improve structural performance by modifying the external geometry.
- The mesh topology remains unchanged.
- Mesh morphing was used: no mesh regeneration required.
- Managed with RBF Morph, which applies continuous and controlled deformations to the mesh using Radial Basis Functions (RBF).

# RBF MORPH SET-UP



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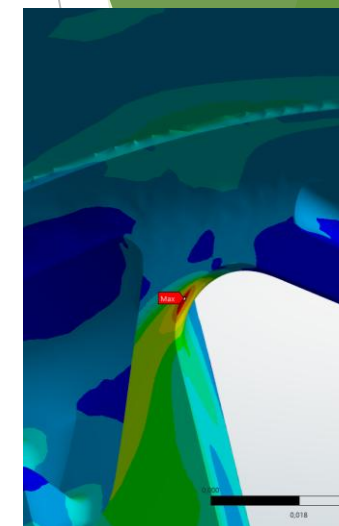
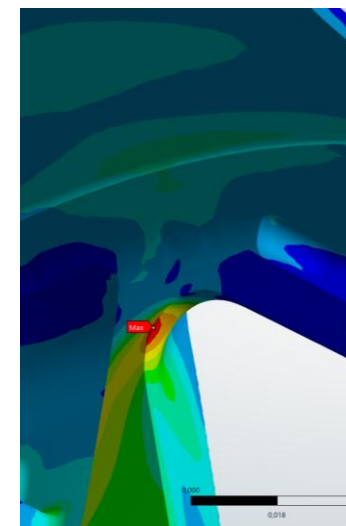


# MASS OPTIMIZATION RESULTS

*Trial and Error*



Parameters	Values
Offset-1	-1,6 mm
Offset-2	-4,5 mm
Offset-3	-4 mm



-3,55%

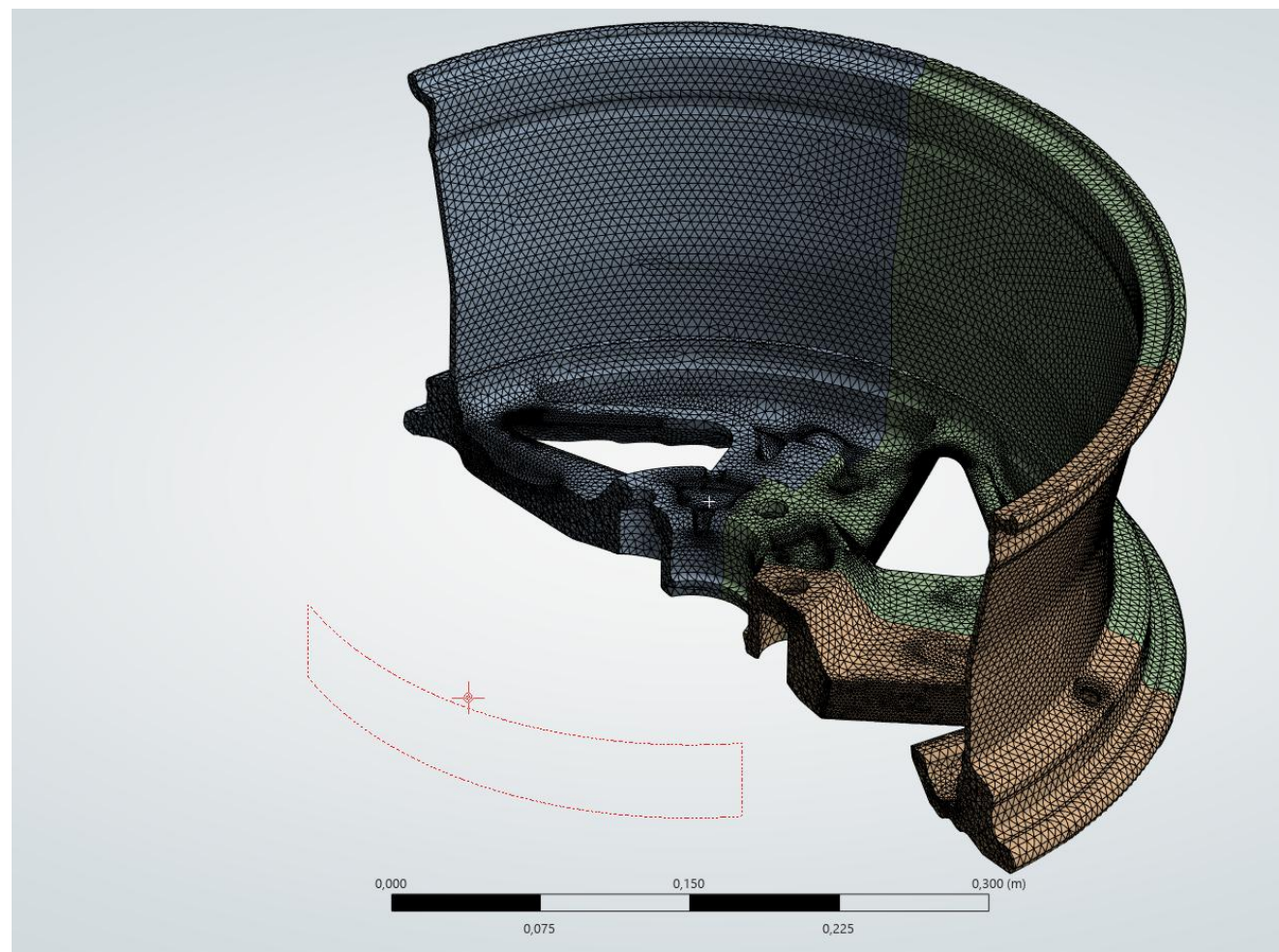
Parameters	Baseline model values	Optimized model values
Mass (kg)	14,1	13,72
Volume (m <sup>3</sup> )	5,22·10 <sup>-3</sup>	5,08·10 <sup>-3</sup>
Maximum Stress (MPa)	81,24	93,10

# MASS OPTIMIZATION RESULTS



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Baseline View



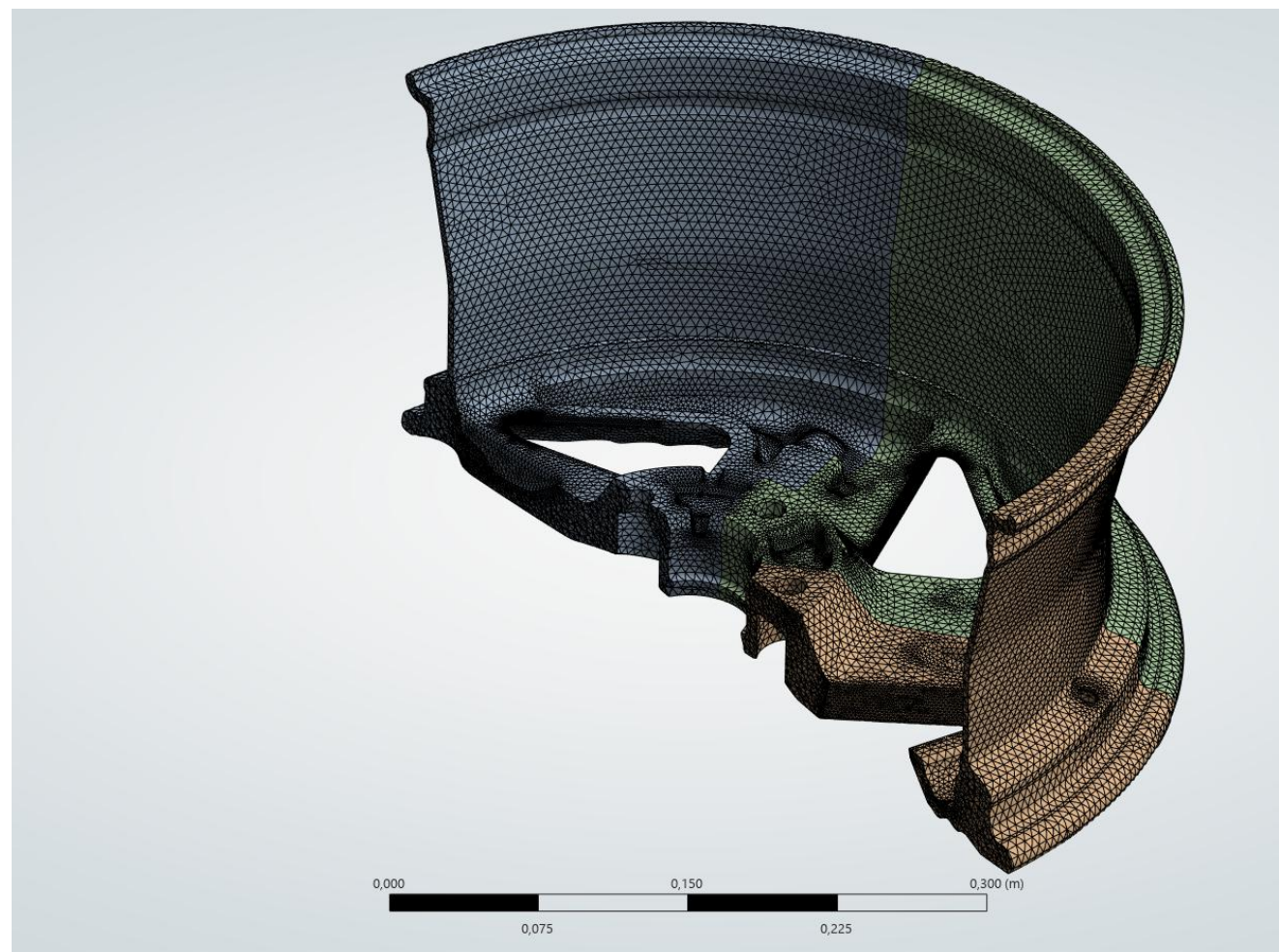


# MASS OPTIMIZATION RESULTS



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Morphed View

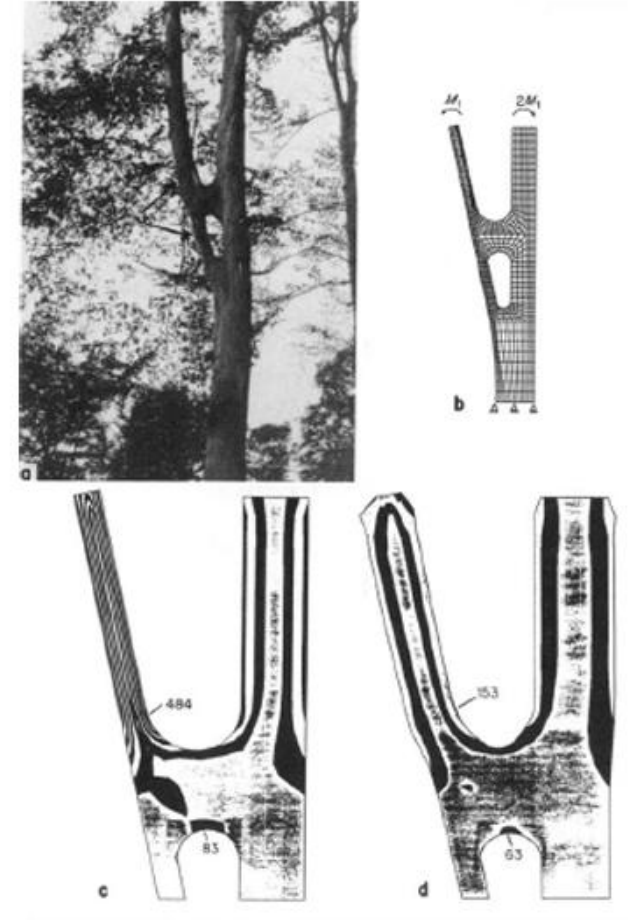


# BIOLOGICAL GROWTH METHOD (BGM)



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- BGM is an optimization method inspired by biological growth mechanisms (bones, trees).
- It aims to achieve a target stress by adding material where needed and removing it where excessive.
- The geometry is modified locally.
- Implemented through RBF Morph, it operates iteratively based on FEM results.



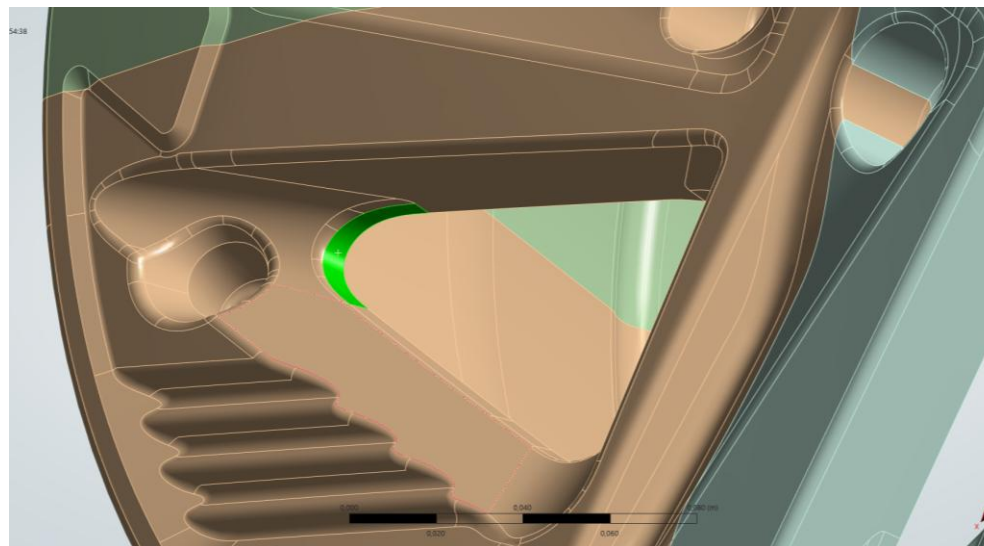
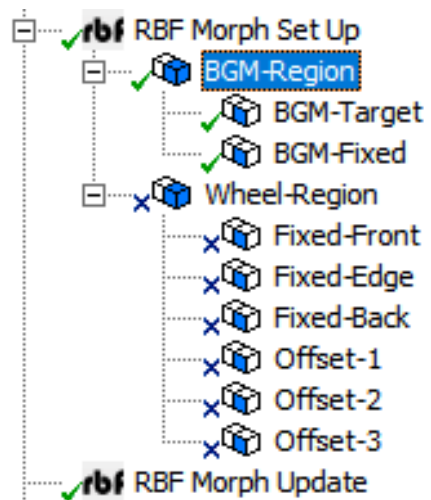
$$\dot{\varepsilon}_{def} = \beta(\sigma(x, y, z) - \sigma_{ref}) \quad \forall x, y, z \in D$$

$$\varepsilon_{def} = \beta(\sigma(x, y, z) - \sigma_{ref}) \Delta t$$

# BGM IN RBF MORPH



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Legge spostamento in RBF  
Morph

$$S_{node} = \frac{\sigma_{node} - \sigma_{th}}{\sigma_{max} - \sigma_{min}} \cdot d$$



80 MPa

- The coordinates of the displaced or fixed nodes are passed to the RBF function.
- The RBF function interpolates a known function at discrete points and updates the new mesh.

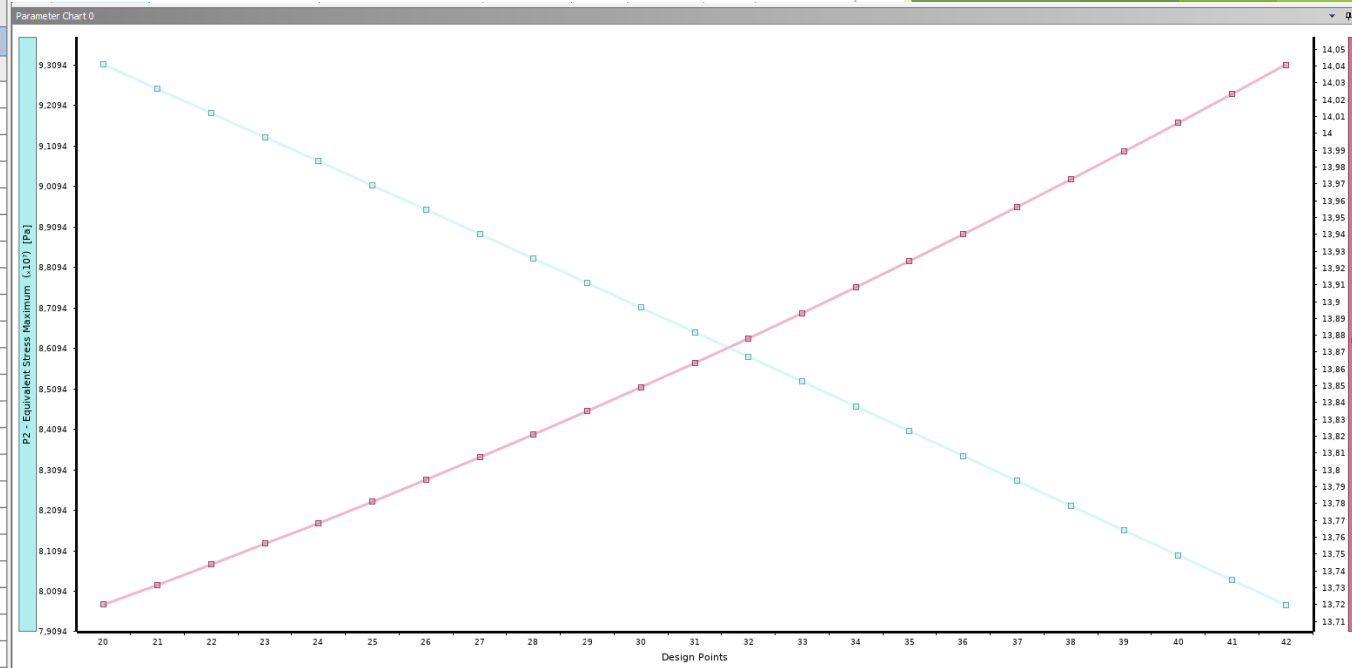


# BGM OPTIMIZATION RESULTS



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	A	B	C	D	E	F	G
1	Name	P1 - RBF Morph Set Up Shape ID	P2 - Equivalent Stress Maximum	P3 - my_Mass	Retain	Retained Data	Note
2	Units		Pa				
3	DP 20	0	9,3098E+07	13,72	<input checked="" type="checkbox"/>	✓	
4	DP 21	1	9,2499E+07	13,732	<input checked="" type="checkbox"/>	✓	
5	DP 22	2	9,1903E+07	13,744	<input checked="" type="checkbox"/>	✓	
6	DP 23	3	9,1304E+07	13,756	<input checked="" type="checkbox"/>	✓	
7	DP 24	4	9,0718E+07	13,768	<input checked="" type="checkbox"/>	✓	
8	DP 25	5	9,0115E+07	13,781	<input checked="" type="checkbox"/>	✓	
9	DP 26	6	8,952E+07	13,794	<input checked="" type="checkbox"/>	✓	
10	DP 27	7	8,8917E+07	13,807	<input checked="" type="checkbox"/>	✓	
11	DP 28	8	8,831E+07	13,821	<input checked="" type="checkbox"/>	✓	
12	DP 29	9	8,7707E+07	13,835	<input checked="" type="checkbox"/>	✓	
13	DP 30	10	8,7097E+07	13,849	<input checked="" type="checkbox"/>	✓	
14	DP 31	11	8,6492E+07	13,863	<input checked="" type="checkbox"/>	✓	
15	DP 32	12	8,5886E+07	13,878	<input checked="" type="checkbox"/>	✓	
16	DP 33	13	8,5275E+07	13,893	<input checked="" type="checkbox"/>	✓	
17	DP 34 (Current)	14	8,466E+07	13,909	<input checked="" type="checkbox"/>	✓	
18	DP 35	15	8,4049E+07	13,924	<input checked="" type="checkbox"/>	✓	
19	DP 36	16	8,3436E+07	13,94	<input checked="" type="checkbox"/>	✓	
20	DP 37	17	8,2815E+07	13,956	<input checked="" type="checkbox"/>	✓	
21	DP 38	18	8,2201E+07	13,973	<input checked="" type="checkbox"/>	✓	
22	DP 39	19	8,159E+07	13,989	<input checked="" type="checkbox"/>	✓	
23	DP 40	20	8,0986E+07	14,006	<input checked="" type="checkbox"/>	✓	
24	DP 41	21	8,037E+07	14,023	<input checked="" type="checkbox"/>	✓	
25	DP 42	22	7,9761E+07	14,041	<input checked="" type="checkbox"/>	✓	
*					<input type="checkbox"/>		



The algorithm will converge to the set target stress, providing a solution at each completed iteration step.



A choice must be made!

# BGM OPTIMIZATION RESULTS

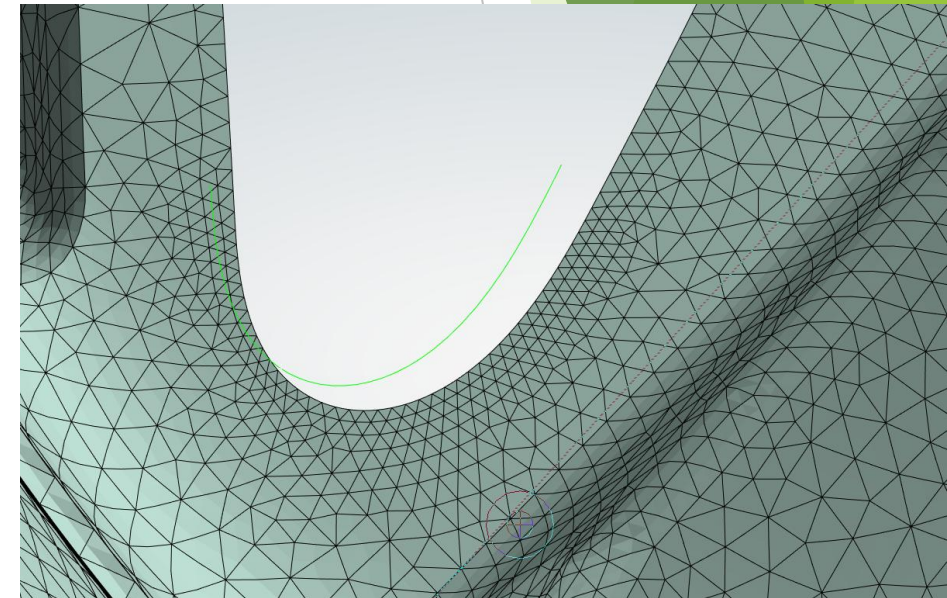


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Configuration	Maximum Stress (MPa)	Mass (Kg)
Baseline	81,24	14,1
Mass-optimized model	93,10	13,72
Selected Configuration (BGM)	84,66	13,91
Last BGM Configuration	79,8	14,04

The selected configuration:

- ✓ reduces the overall mass by 1.35% compared to the initial model.
- ✓ keeps the increase in maximum stress within 4% compared to the initial model.



# CONCLUSIONS



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- Mass reduction of 1.5 kg;
- Stress redistribution;
- Multiple solutions, all implementable depending on the specific requirements and the relevant industrial context;
- Preservation of the design style.

Matteo Bisin



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